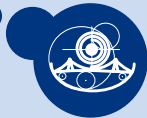


# Damping optimization in mechanical systems using parametric model reduction



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[SEMINAR ZA PRIM. MAT. I TEORIJU UPRAVLJANJA, UNIDU]

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## 1 Introduction

Problem formulation

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## 2 Parametric model reduction

Basic structure

ROM

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## 3 Numerical experiments

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Thermal Model

Damping example



We consider vibrational system

$$M\ddot{q}(t) + \overbrace{(C_{int} + B_2GB_2^T)}^{C(g)=\text{damping part}}\dot{q}(t) + Kq(t) = E_2w(t),$$
$$y(t) = H_1q(t).$$

- $M, K \in \mathbb{R}^{n \times n}$  mass and stiffness, the symmetric and positive definite
- $q \in \mathbb{R}^n$  state vector and  $y$  is output vector determined by  $H_1 \in \mathbb{R}^{\ell \times n}$ ,
- $E_2 \in \mathbb{R}^{n \times m}$  determines primary excitation matrix and vector  $w \in \mathbb{R}^m$  corresponds to primary excitation input.
- $C_{int} \in \mathbb{R}^{n \times n}$  internal damping e.g.  $C_{int} = \alpha_c C_{crit}$ , where

$$C_{crit} = 2M^{1/2}\sqrt{M^{-1/2}KM^{-1/2}}M^{1/2},$$

- $G = \text{diag}(g_1, g_2, \dots, g_k)$ ,  $g_i \geq 0$  damping coefficients.



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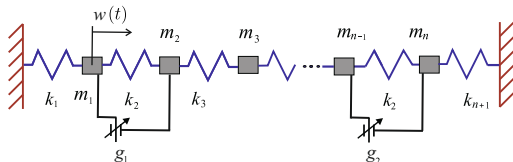
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## Example: *n*-mass oscillator or oscillator ladder

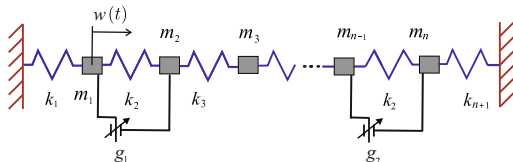


$$M = \text{diag}(m_1, m_2, \dots, m_n), \quad C(g) = \alpha_c C_{crit} + B_2 G B_2^T, \\
B_2 G B_2^T = g_1 (e_i - e_{i+1})(e_i - e_{i+1})^T + g_2 (e_j - e_{j+1})(e_j - e_{j+1})^T.$$

$$K = \begin{pmatrix} k_1 + k_2 & -k_2 & & & \\ -k_2 & k_2 + k_3 & -k_3 & & \\ & \ddots & \ddots & \ddots & \\ & & -k_{n-1} & k_{n-1} + k_n & -k_n \\ & & & -k_n & k_n + k_{n+1} \end{pmatrix}.$$



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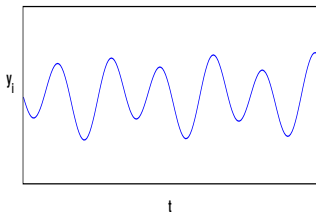
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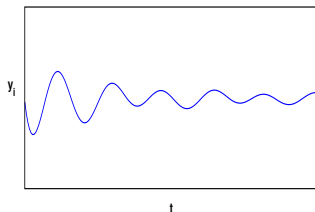


The principal goal is to determine an optimal damping matrix that will minimize the influence of the input  $w$  (viewed as a disturbance) on the output,  $y$ .

$(M, K)$



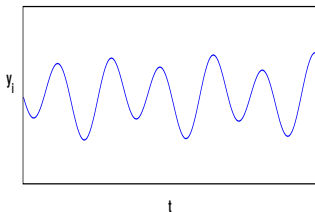
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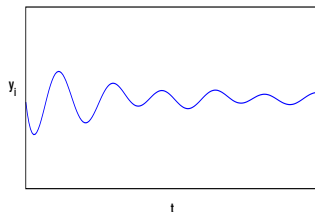


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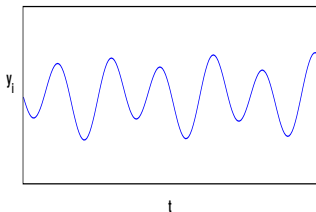




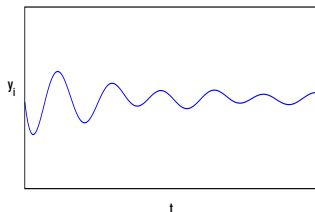


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## Linearization

With the substitutions  $x_1(t) := q(t)$ ,  $x_2(t) := \dot{q}(t)$  and  $x := \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$  we obtain a first-order representation of the closed-loop system

$$\underbrace{\begin{bmatrix} I_n & 0 \\ 0 & M \end{bmatrix}}_{\mathcal{E}} \dot{x}(t) = \underbrace{\begin{bmatrix} 0 & I_n \\ -K & -C(g) \end{bmatrix}}_A x(t) + \underbrace{\begin{bmatrix} 0 \\ E_2 \end{bmatrix}}_B w(t),$$

$$y(t) = \underbrace{\begin{bmatrix} H_1 & 0 \end{bmatrix}}_C x(t).$$

Using the Laplace transform we obtain the closed-loop transfer function

$$\begin{aligned} \mathbf{H}(g, s) &= H_1 (s^2 M + sC(g) + K)^{-1} E_2 \\ &= [H_1 \quad 0] \left( s \begin{bmatrix} I_n & 0 \\ 0 & M \end{bmatrix} - \begin{bmatrix} 0 & I_n \\ -K & -C(g) \end{bmatrix} \right)^{-1} \begin{bmatrix} 0 \\ E_2 \end{bmatrix} \end{aligned}$$



## $\mathcal{H}_2$ norm of a system

Define the space

$$\mathcal{H}_2^{\ell \times m} := \left\{ \mathbf{H} : \mathbb{C}^+ \rightarrow \mathbb{C}^{\ell \times m} \mid \mathbf{H} \text{ is analytic in } \mathbb{C}^+ \text{ and } \int_{-\infty}^{+\infty} \text{tr}(\mathbf{H}(i\omega)^* \mathbf{H}(i\omega)) d\omega < \infty \right\},$$

$$\|\mathbf{H}(g, \cdot)\|_{\mathcal{H}_2} = \left( \frac{1}{2\pi} \int_{-\infty}^{+\infty} \text{tr}(\mathbf{H}(g, i\omega)^* \mathbf{H}(g, i\omega)) d\omega \right)^{\frac{1}{2}}.$$

It can be expressed via the solution of a Lyapunov equation, i.e.

$$\|\mathbf{H}(g, \cdot)\|_{\mathcal{H}_2} = \left( \frac{1}{2\pi} \text{tr} B^T \mathcal{P} B \right)^{\frac{1}{2}}, \quad \text{where} \quad A^T \mathcal{P} + \mathcal{P} A = -C^T C$$

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[T./Beattie/Gugercin18, Benner/Kurschner/T./Truhar16]



$\mathcal{H}_\infty$  norm of a system <sup>1</sup>

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$$\|\mathbf{H}(g, \cdot)\|_{\mathcal{H}_\infty} := \sup_{\lambda \in \mathbb{C}^+} \|\mathbf{H}(g, \lambda)\|_2 = \sup_{\omega \in \mathbb{R}} \|\mathbf{H}(g, i\omega)\|_2.$$

One can also consider certain mixed performance measures:

- $\|\mathbf{H}(g, \cdot)\|_{\mathcal{H}_\infty/\mathcal{H}_2}$
- criterion that combines  $\|\mathbf{H}(g, \cdot)\|_{\mathcal{H}_2}$  and total average energy <sup>2</sup>

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<sup>1</sup>[T./Voigt20]

<sup>2</sup>[Nakić/T./Truhar19]



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## Main drawback of direct methods

Damping optimization (position and viscosity optimization):

In the  $n$ -mass oscillator:

$$C_{ext} = g(e_i - e_{i+1})(e_i - e_{i+1})^T + g(e_j - e_{j+1})(e_j - e_{j+1})^T,$$

there is a problem with determining optimal  $(i, j)$ ,  $1 \leq i \leq j \leq n$  and  $g$ .

For example if  $n = 1000$ :

discrete optimization over 500 000 different damping positions.

Efficient overall algorithm for optimization of damping positions is still needed !

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Consider a parametric LTI dynamical systems represented as

$$\begin{aligned} E\dot{x}(t; p) &= A(p)x(t; p) + Bu(t), \\ y(t; p) &= Cx(t; p), \end{aligned}$$

where  $E, A(p) \in \mathbb{R}^{n \times n}$ ,  $B \in \mathbb{R}^{n \times m}$  and  $C \in \mathbb{R}^{l \times n}$ .

- $x(t; p) \in \mathbb{R}^n$  denotes the state variable
- $u(t) \in \mathbb{R}^m$  and  $y(t; p) \in \mathbb{R}^l$  represent the inputs and outputs of the system, resp.

We will denote this system with  $[E, A(p), B, C]$ .



For parameter  $p$  we can approximate our system with reduced system <sup>1</sup>

$$\begin{aligned} E_r \dot{x}_r(t; p) &= A_r(p) x_r(t; p) + B_r u(t; p), \\ y_r(t; p) &= C_r x_r(t; p), \end{aligned}$$

where matrices  $V_r \in \mathbb{R}^{n \times r}$  and  $W_r \in \mathbb{R}^{n \times r}$  determine reduced system

$$\begin{aligned} E_r &= (W_r)^T E V_r, & A_r &= (W_r)^T A V_r, \\ B_r &= (W_r)^T B & \text{and} & \quad C_r = C V_r. \end{aligned}$$

For set of sampling parameters  $p^1, \dots, p^s$  one can calculate truncation matrices and for global basis we can construct truncation matrices by  $V = [V_r^1, \dots, V_r^s]$  and  $W = [W_r^1, \dots, W_r^s]$ .

Problem: reduced order model depends on sampling parameters, but also which sampling one should use .

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<sup>1</sup>[Benner/Cohen/Ohlberger/Willcox2017], [Benner/Gugercin/Willcox2015],  
[Quarteroni/Manzoni/Negri2016], [Quarteroni/Rozza/Manzoni2011]



We would like to remove the need for parametric sampling, which requires identifying particular parameters of interest!

We consider system where  $A(p)$  depends on  $k \ll n$  parameters  $p = (p_1, p_2, \dots, p_k)$  such that we may write

$$A(p) = A_0 + U \operatorname{diag}(p_1, p_2, \dots, p_k) V^T = A_0 + \sum_{i=1}^k p_i u_i v_i,$$

where  $U, V \in \mathbb{R}^{n \times k}$  are fixed.

Full-order transfer function

$$\mathbf{H}(s; p) = C(sE - A(p))^{-1}B.$$

Aim: to produce a ROM that retains the structure of parametric dependence and offers uniformly high fidelity across the full parameter range.



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## Structure in damping example.

By defining the state-vector  $x = [q^T \dot{q}^T]^T$  we obtain:

$$\begin{bmatrix} I & 0 \\ 0 & M \end{bmatrix} \dot{x}(t) = A(p)x(t) + \begin{bmatrix} 0 \\ E_2 \end{bmatrix} w(t),$$

$$z(t) = \begin{bmatrix} H_1 & 0 \end{bmatrix} x(t), \quad \text{where}$$

$$A(p) = \begin{bmatrix} 0 & I \\ -K & -C_{int} \end{bmatrix} + \begin{bmatrix} 0 \\ B_2 \end{bmatrix} \text{diag}(p_1, p_2, \dots, p_k) \begin{bmatrix} 0 & B_2^T \end{bmatrix}.$$

## Further extensions to the cases with higher rank.

E.g.  $A(p) = A_0 + p_1 A_1 + p_2 A_2$  where both  $A_1, A_2$  have rank-2.

Then, one can write  $A_1 = [u_1 \ u_2][v_1 \ v_2]^T$  and  $A_2 = [u_3 \ u_4][v_3 \ v_4]^T$ .

With  $U = [u_1 \ u_2 \ u_3 \ u_4]$  and  $V = [v_1 \ v_2 \ v_3 \ v_4]$  we obtain

$$A(p) = A_0 + p_1 A_1 + p_2 A_2 = A_0 + U \text{diag}(p_1, p_1, p_2, p_2) V^T.$$



## The key observation!

$$\mathbf{H}(s; p) = C \left( \hat{A}(s) - U \operatorname{diag}(p_1, p_2, \dots, p_k) V^T \right)^{-1} B, \quad \hat{A}(s) = sE - A_0.$$

We use the Sherman-Morrison-Woodbury formula.

$$\mathbf{H}(s; p) = \mathbf{H}_1(s) - \mathbf{H}_2(s) D(p) (I_k + D(p) \mathbf{H}_3(s) D(p))^{-1} D(p) \mathbf{H}_4(s),$$

where parameters are encoded in diagonal matrix

$$D(p) = \operatorname{diag}(\sqrt{p_1}, \sqrt{p_2}, \dots, \sqrt{p_k}) \text{ and}$$

$$\begin{aligned} \mathbf{H}_1(s) &= C \hat{A}(s)^{-1} B, & \mathbf{H}_2(s) &= C \hat{A}(s)^{-1} U, \\ \mathbf{H}_3(s) &= V^T \hat{A}(s)^{-1} U, & \mathbf{H}_4(s) &= V^T \hat{A}(s)^{-1} B. \end{aligned}$$

We construct a parameterized reduced order model by using four subsystems which **do not depend on parameters**:

$$[E, A_0, B, C], [E, A_0, U, V^T], [E, A_0, U, C], \text{ and } [E, A_0, B, V^T].$$



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## Approach 1: Reduced model based on vector fitting approach

### Offline steps

- For the predetermined points in the complex plane  $\xi_1, \dots, \xi_N$  calculate

$$\mathbf{H}_1(\xi_i), \mathbf{H}_2(\xi_i), \mathbf{H}_3(\xi_i), \mathbf{H}_4(\xi_i) \quad \text{for } i = 1, \dots, N.$$

These samples do not depend on parameters!

### Online steps

- For any given parameter  $\mathbf{p} = (p_1, p_2, \dots, p_k)$  calculate  $\mathbf{H}(\xi_i; \mathbf{p})$  for  $i = 1, \dots, N$  using obtained formula.
- Based on  $\mathbf{H}(\xi_1; \mathbf{p}), \dots, \mathbf{H}(\xi_N; \mathbf{p})$  obtain reduced system with transfer function  $\hat{\mathbf{H}}(s; \mathbf{p})$  using vector fitting approach.

The quality of approximations is determined by

$$e(\mathbf{H}(\cdot; \mathbf{p}), \hat{\mathbf{H}}(\cdot; \mathbf{p})) = \sum_{i=1}^N \left\| \mathbf{H}(\xi_i; \mathbf{p}) - \hat{\mathbf{H}}(\xi_i; \mathbf{p}) \right\|_F^2 / \sum_{i=1}^N \left\| \mathbf{H}(\xi_i; \mathbf{p}) \right\|_F^2.$$





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## Approach 2: ROM based on reduction of subsystems

### Offline steps

- For underlying subsystems calculate reduced systems using model reduction techniques for non-parametric systems

$$[E, A_0, B, C] \rightarrow \hat{\mathbf{H}}_1(s), \text{ using order } r_1;$$

$$[E, A_0, U, V^T] \rightarrow \hat{\mathbf{H}}_2(s), \text{ using order } r_2;$$

$$[E, A_0, U, C] \rightarrow \hat{\mathbf{H}}_3(s), \text{ using order } r_3;$$

$$[E, A_0, B, V^T] \rightarrow \hat{\mathbf{H}}_4(s), \text{ using order } r_4;$$

e. g. using balanced truncation or IRKA approach.

### Online steps

- For any given parameter  $\mathbf{p} = (p_1, p_2, \dots, p_k)$  obtain approximated system  $\hat{\mathbf{H}}(s; \mathbf{p})$  by

$$\mathbf{H}(s; \mathbf{p}) \approx \hat{\mathbf{H}}_1(s) - \hat{\mathbf{H}}_2(s)D(\mathbf{p})(I + D(\mathbf{p})\hat{\mathbf{H}}_3(s)D(\mathbf{p}))^{-1}D(\mathbf{p})\hat{\mathbf{H}}_4(s)$$



## Uniform stability of the parameterized reduced model

### Theorem

*Suppose that the full parameterized model  $\mathbf{H}(s, p)$  has been decomposed into subsystems  $\mathbf{H}_1(s)$ ,  $\mathbf{H}_2(s)$ , and  $\mathbf{H}_4(s)$  that are each asymptotically stable, and a subsystem  $\mathbf{H}_3(s)$  that is positive real. If the corresponding reduced subsystems  $\hat{\mathbf{H}}_1(s)$ ,  $\hat{\mathbf{H}}_2(s)$ , and  $\hat{\mathbf{H}}_4(s)$  retain asymptotic stability, and  $\hat{\mathbf{H}}_3(s)$  retains positive-realness, then the reduced parameterized model  $\hat{\mathbf{H}}(s, p)$  in*

$$\hat{\mathbf{H}}(s; p) = \hat{\mathbf{H}}_1(s) - \hat{\mathbf{H}}_2(s)D(p)(I + D(p)\hat{\mathbf{H}}_3(s)D(p))^{-1}D(p)\hat{\mathbf{H}}_4(s).$$

*is uniformly asymptotically stable for nonnegative parameters encoded in  $p$ .*



In order to calculate error bound we consider full order transfer function

$$\mathbf{H}(s; p) = \mathbf{H}_1(s) - \mathbf{H}_2(s)D(p)(I_k + D(p)\mathbf{H}_3(s)D(p))^{-1}D(p)\mathbf{H}_4(s),$$

and corresponding reduced order transfer function

$$\hat{\mathbf{H}}(s; p) = \hat{\mathbf{H}}_1(s) - \hat{\mathbf{H}}_2(s)D(p)(I_k + D(p)\hat{\mathbf{H}}_3(s)D(p))^{-1}D(p)\hat{\mathbf{H}}_4(s),$$

we would like to have upper bound for the error

$$\|\mathbf{H}(\cdot; p) - \hat{\mathbf{H}}(\cdot; p)\| \leq ?$$



## Error bound

It can be shown that

$$\begin{aligned} \mathbf{H}(\cdot; p) - \hat{\mathbf{H}}(\cdot; p) &= [\mathbf{H}_1 - \hat{\mathbf{H}}_1] + [\hat{\mathbf{H}}_2 - \mathbf{H}_2]D(p)(I + D(p)\hat{\mathbf{H}}_3D(p))^{-1}D(p)\hat{\mathbf{H}}_4 + \\ &\quad + \mathbf{H}_2D(p)(I + D(p)\hat{\mathbf{H}}_3D(p))^{-1}D(p)[\hat{\mathbf{H}}_4 - \mathbf{H}_4] + \\ &\quad + \mathbf{H}_2D(p)(I + D(p)\hat{\mathbf{H}}_3D(p))^{-1}D(p)[\mathbf{H}_3 - \hat{\mathbf{H}}_3]D(p)(I + D(p)\mathbf{H}_3D(p))^{-1}D(p)\hat{\mathbf{H}}_4 \end{aligned}$$

Thus, we have

$$\begin{aligned} \|\mathbf{H}(\cdot; p) - \hat{\mathbf{H}}(\cdot; p)\| &\leq \|\mathbf{H}_1 - \hat{\mathbf{H}}_1\| + \|\hat{\mathbf{H}}_2 - \mathbf{H}_2\| \|D(p)(I + D(p)\hat{\mathbf{H}}_3D(p))^{-1}D(p)\hat{\mathbf{H}}_4\| + \\ &\quad + \|\mathbf{H}_2D(p)(I + D(p)\mathbf{H}_3D(p))^{-1}D(p)[\hat{\mathbf{H}}_4 - \mathbf{H}_4]\| + \\ &\quad + \|\mathbf{H}_2D(p)(I + D(p)\hat{\mathbf{H}}_3D(p))^{-1}D(p)\| \|\mathbf{H}_3 - \hat{\mathbf{H}}_3\| \|D(p)(I + D(p)\mathbf{H}_3D(p))^{-1}D(p)\hat{\mathbf{H}}_4\| \end{aligned}$$



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Which means that we have a bound in terms of

$$\|\mathbf{H}(\cdot; p) - \hat{\mathbf{H}}(\cdot; p)\| \lesssim \overbrace{\varepsilon_1 + \varepsilon_2 f_1(p, \hat{\mathbf{H}}_3, \hat{\mathbf{H}}_4) + \varepsilon_4 f_2(p, \hat{\mathbf{H}}_2, \hat{\mathbf{H}}_3) + \varepsilon_3 f_3(p, \hat{\mathbf{H}}_2, \hat{\mathbf{H}}_3, \hat{\mathbf{H}}_4)}^{f(p)}$$



## Surrogate optimization with reduced parametric models

A major cost in parameter optimization is the repeated evaluation of the  $\mathcal{H}_2$  norm.

We can use the approach 1 or 2 to accelerate computational cost, so we solve a surrogate optimization problem

$$\hat{p}^* = \arg \min_{p \in \Omega} \left\| \hat{\mathbf{H}}(\cdot, p) \right\|_{\mathcal{H}_2},$$

where the reduced parametric transfer function  $\hat{\mathbf{H}}(\cdot, p)$  will be constructed using either approach 1 or approach 2, without need for parameter sampling.

Assume  $p^*$  is the minimizer and note that

$$\left\| \mathbf{H}(\cdot, p^*) \right\|_{\mathcal{H}_2} \leq \left\| \mathbf{H}(\cdot, p^*) - \hat{\mathbf{H}}(\cdot, p^*) \right\|_{\mathcal{H}_2} + \left\| \hat{\mathbf{H}}(\cdot, p^*) \right\|_{\mathcal{H}_2}.$$

The surrogate optimization problem will minimize the second term.



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## Parameter optimization using reduced models via approach 1

- 1: Choose the reduced order so that  $e(p^0) < \tau$ .
- 2: Solve the surrogate optimization problem

$$\hat{p}^* = \arg \min_p \left\| \hat{\mathbf{H}}(\cdot, p) \right\|_{\mathcal{H}_2}$$

with the initial guess  $p^0$  and VF approach for  $\hat{\mathbf{H}}_p$ , using  $\{\mathbf{H}_i(\xi_i)\}_{i=1}^N$ .

- 3: **while** minimizer  $p^*$  such that  $e(p^*) > \tau$  **do**
- 4:  $p^0 = \hat{p}^*$
- 5: Increase the reduced order so that  $e(\hat{p}^*) < \tau$ .
- 6: Determine the new minimizer by solving the

$$\hat{p}^* = \arg \min_p \left\| \hat{\mathbf{H}}(\cdot, p) \right\|_{\mathcal{H}_2}$$

using the updated  $\hat{\mathbf{H}}$ , the initial guess  $p^0$ , and tolerance  $\nu$ .

- 7: **end while**



## Parameter optimization using reduced models via approach 2

- 1: Choose the reduced orders  $r_1, r_2, r_3, r_4$  (and  $\hat{\mathbf{H}}_1, \hat{\mathbf{H}}_2, \hat{\mathbf{H}}_3, \hat{\mathbf{H}}_4$ ) so that  $f(\mathbf{p}^0) < \tau \|\hat{\mathbf{H}}(\cdot, \mathbf{p}^0)\|_{\mathcal{H}_2}$ .
- 2: Solve the surrogate optimization problem

$$\hat{\mathbf{p}}^* = \arg \min_{\mathbf{p}} \left\| \hat{\mathbf{H}}(\cdot, \mathbf{p}) \right\|_{\mathcal{H}_2}$$

with the initial guess  $\mathbf{p}^0$  and tolerance  $\nu$ .

- 3: **while** minimizer  $\mathbf{p}^*$  such that  $f(\mathbf{p}^*) > \tau \|\hat{\mathbf{H}}(\cdot, \mathbf{p}^0)\|_{\mathcal{H}_2}$  **do**
- 4:    $\mathbf{p}^0 = \hat{\mathbf{p}}^*$
- 5:   Increase the orders  $r_1, r_2, r_3, r_4$  s.t.  $f(\hat{\mathbf{p}}^*) < \tau \|\hat{\mathbf{H}}(\cdot, \mathbf{p}^0)\|_{\mathcal{H}_2}$ .
- 6:   Determine the new minimizer by solving the
$$\hat{\mathbf{p}}^* = \arg \min_{\mathbf{p}} \left\| \hat{\mathbf{H}}(\cdot, \mathbf{p}) \right\|_{\mathcal{H}_2}$$
using the updated  $\hat{\mathbf{H}}$ , the initial guess  $\mathbf{p}^0$ , and tolerance  $\nu$ .
- 7: **end while**



We consider example from [Penzl 1999]. The full-order system is known and defined by state-space matrices

$$A = \text{diag}(A_1(p_1), A_2(p_2), A_3(p_3), -1, -2, \dots, -N)$$
$$A_i(p_i) = \begin{bmatrix} -1 & p_i \\ -p_i & -1 \end{bmatrix}, \quad \text{for } i = 1, \dots, 3$$

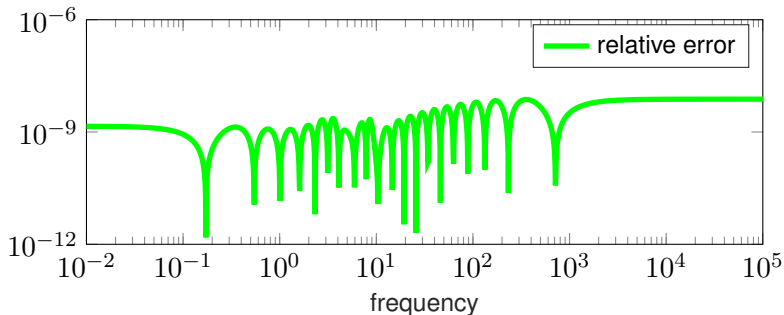
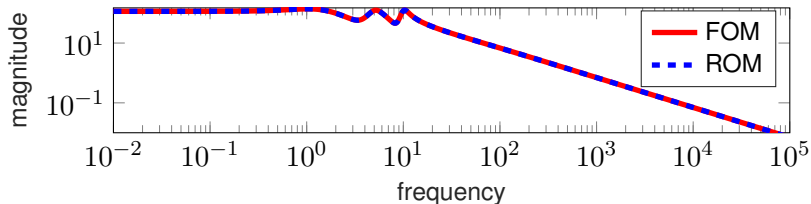
Matrix  $C \in \mathbb{R}^{1 \times (N+6)}$  where

$$c_i = \begin{cases} 10, & i = 1, \dots, 6, \\ 1, & i = 7, \dots, N. \end{cases}$$

$B = C^T$  and number of states  $N = 100$ . The parameters  $p_1, p_2, p_3$  represent the imaginary part of the two eigenvalues of the diagonal block  $A_i(p_i)$ , respectively. Here we use that  $p_2 = 5p_1$  and  $p_3 = 20p_1$ . We illustrate approach based on balanced truncation of subsystems where four underlying subsystems were reduced to dimensions 10, 1, 6, 1.



$p=(1.00, 5.00, 20.00)$







We consider thermal conduction in a semiconductor chip from Oberwolfach Benchmark Collection.

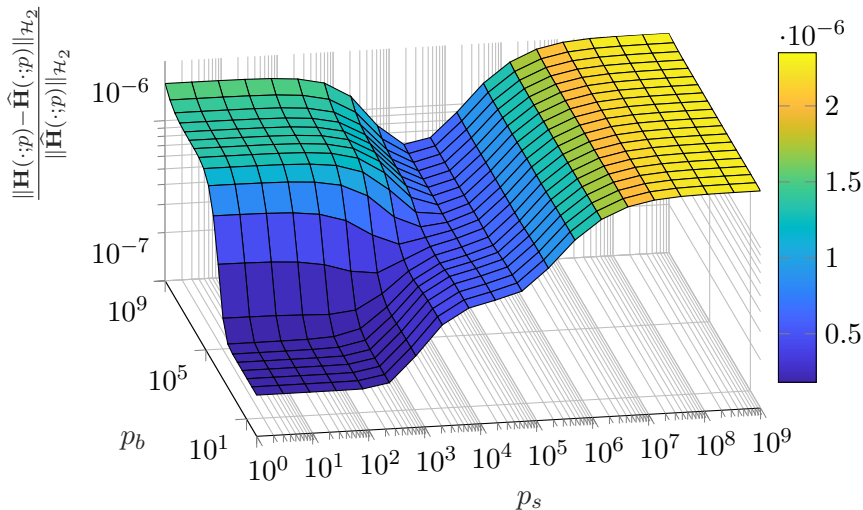
The full-order system is:

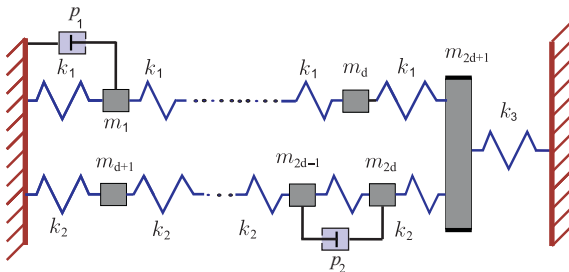
$$E\dot{x} = (A - p_t A_t - p_b A_b - p_s A_s)x + Bu$$
$$y = Cx, \quad \text{where}$$

- $E \in \mathbb{R}^{4257 \times 4257}$  corresponds to heat capacity and  $A$  to heat conductivity matrix
- $B \in \mathbb{R}^{1 \times 4257}$  is the load vector and  $C \in \mathbb{R}^{7 \times 4257}$
- $A_t$ ,  $A_b$  and  $A_s$  are the diagonal matrices from the discretization of the convection boundary conditions with ranks 111, 99 and 31, resp.
- Parameters  $p_t, p_b, p_s$  represent film coefficients.



We fix  $p_t = 1000$  and vary both  $p_b$  and  $p_s$  between 1 and  $10^9$ .  
 Reduced dim. of subsystems:  $r_1 = 46, r_2 = 66, r_3 = 200, r_4 = 16$ .





The mass and the stiffness matrix are given by

$$K = \begin{bmatrix} K_{11} & & -\kappa_1 \\ & K_{22} & -\kappa_2 \\ -\kappa_1^T & -\kappa_2^T & k_1 + k_2 + k_3 \end{bmatrix}, \quad K_{ii} = k_i \begin{bmatrix} 2 & -1 & & & \\ -1 & 2 & -1 & & \\ & \ddots & \ddots & \ddots & \\ & & -1 & 2 & -1 \\ & & & -1 & 2 \end{bmatrix},$$

$$\kappa_i = [0 \quad \dots \quad 0 \quad k_i] \text{ for } i = 1, 2 \text{ and } M = \text{diag}(m_1, m_2, \dots, m_n).$$





$d = 900 \Rightarrow n = 1801$ , with  $m_{1801} = 1000$  and

$$m_i = \begin{cases} 1000 - \frac{i}{2}, & i = 1, \dots, 450, \\ i + 325, & i = 451, \dots, 900, \\ 1300 - \frac{i}{4}, & i = 901, \dots, n. \end{cases}$$

The stiffness values are given by

$$k_1 = 500, k_2 = 200, k_3 = 300.$$

The primary excitation are 5 disturbances applied to the 4 masses closest to the left-hand side and one mass closest to the right-hand side of oscillator.

We are interested in 2 displacements, i. e.

$$z(t; p) = [q_{400}(t; p) \quad q_{1300}(t; p)]^T.$$



Internal damping is a small multiple of critical damping

$$C_{int} = 0.04 \cdot M^{1/2} \left( M^{-1/2} K M^{-1/2} \right)^{1/2} M^{1/2}.$$

We consider four dampers with gains  $p_1, p_2, p_3$  and  $p_4$  where geometry of positions is given by

$$B_2 = [e_{j_1} - e_{j_1+10}, e_{j_2}, e_{j_3}, e_{j_3} - e_{j_3+100}],$$

with  $j_1 \in \{100, 300, 500, 700\}$ ,  $j_2 \in \{150, 350, 550, 750\}$ ,  
 $j_3 \in \{1400, 1700\} \Rightarrow 32$  different damping configurations at which  
 $\|\cdot\|_{\mathcal{H}_2}$  norm was minimized.

Gains were optimized with starting point  $p^0 = (100, 100, 100, 100)$  using the full-order model and using proposed reduced systems.



In the approach based on **balanced truncation of subsystems**:

- in all damping configurations, starting reduced dimensions of four subsystems were 280, 300, 480, 430, resp.

In the approach based on **vector fitting approach**

- initial points  $\xi_i, i = 1, \dots, N$ , for  $N = 500$  depending on modally damped system.
- 130 initial poles (chosen using from dominant poles).

The stoping tolerance for parameter optimization was 0.005.

### Time ratio

In average case for one optimization of parameters, new approach **was faster**:

- $\approx 7.8$  times, with usage of reduced model based on balanced truncation of subsystems,
- $\approx 60$  times, with usage of reduced model based on vector fitting approach.



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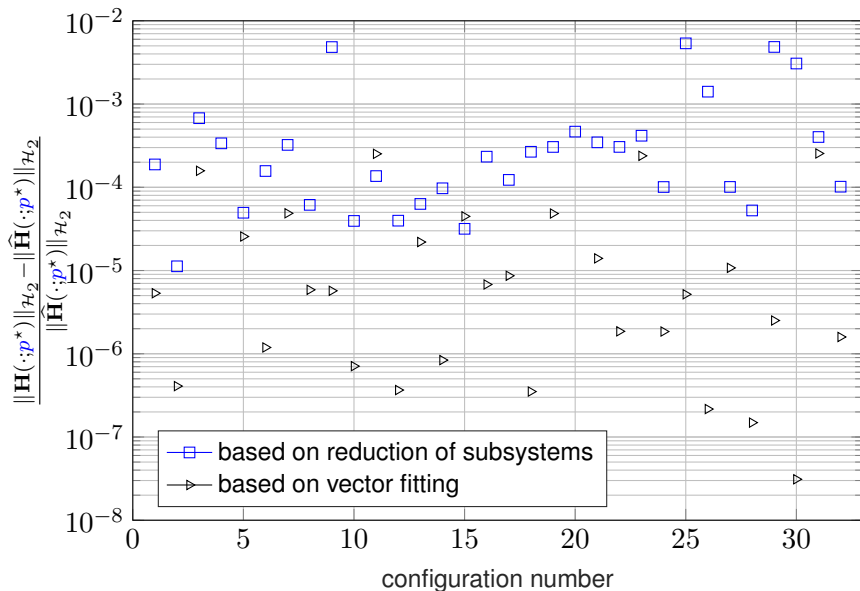
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## Conclusion :

- We have introduced a framework for producing reduced order models of dynamical systems having an affine, low-rank parametric structure.
  - Approach 1: Reduced model based on vector fitting approach.
  - Approach 2: ROM based on reduction of subsystems.
- The new framework does not require any sampling in the parameter domain and instead parametrically combines intermediate subsystems that are nonparametric.
- Can guarantee uniform stability of the aggregated reduced model across the entire parameter domain in many cases.
- These approaches can be deployed efficiently in parameter optimization problems as well.



*Thank you for your attention!*