

# Mixed control of vibrational systems

Ivica Nakić

Department of Mathematics, University of Zagreb  
(joint work with Z. Tomljanović, N. Truhar)

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# Setting

Linear vibrational system, modeled as a 2nd order matrix differential equation

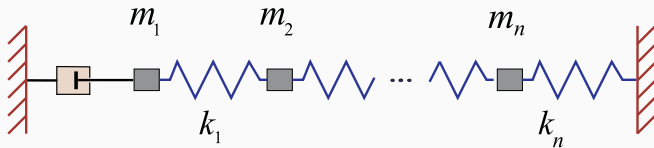
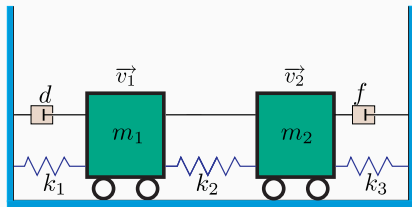
$$M\ddot{x} + C\dot{x} + Kx = F$$

$M$  mass matrix

$C$  damping matrix

$K$  stiffness matrix

$F$  external force



Attenuate unwanted vibrations of the system by the use of passive damping.

In other words, find an appropriate damping matrix  $C$  such that the system vibrates as little as possible.

System will have  $N$  modes of vibration,  $N$  dimension of the system, not all modes are dangerous. Usually there is a range of dangerous ones.

# Why?





# Why?



# Does it work?



Can one successfully attenuate dangerous vibration by passive damping?

Yes!

# How to do it?

Pose the problem of find good/optimal damping mechanism as an **optimization problem**. So we need an optimization **criterion**.

(Too) many criteria in use for different applications (Google Scholar has 342 000 hits for "damping criteria").

# How to do it?

Important classes:

- based on the analysis of stationary system (external force  $F = 0$ , excitation by the initial condition), some interesting ones:
  - based on eigenvalues (e.g.  $\max \Re \lambda$ ,  $\max \frac{\Re \lambda}{|\lambda|}$ )
  - based on the total energy (e.g.  $\max \int_0^\infty E(t) dt$ , avg.  $\int_0^\infty E(t) dt$ )
- based on the analysis of excitation by a particular external force
  - harmonic excitation
  - periodic non-harmonic excitation

# How to do it?

For a random external force we can use the machinery of **control theory**.

Again different criteria, most usefull ones:

→  $H_2$  norm

→  $H_\infty$  norm

$H_2$  norm criterion: external force modeled by (white/coloured) noise, we obtain best damping for a "typical" external force.

$H_2$  norm criterion seems like the best choice for a large class of vibrational systems (non-critical systems, where external environment changes).

When doing numerics, for some configurations we obtain that the damping coefficient should be as large as possible.

This does not make any sense.

It is not a numerics glitch. Can be shown analytically.  
What is going on?

## Toy (academic) example

Let's say we can choose any damping matrix  $D$  we want.

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We are missing the initial condition, control theory setting does not see it.

A photograph of two construction workers in orange safety vests and blue jeans. They are using long-handled tools to guide a large, vertical chute of concrete being poured onto a prepared surface. The concrete is thick and grey. In the background, the wheels of a truck are visible. The scene is outdoors on a construction site.

**If you pour concrete  
over your structure,  
it surely will not  
vibrate.**

Use a mixed criterion, for example

$$\text{avg.} \int_0^\infty E(t) dt + \|\cdot\|_{H_2}$$

We mix one criterion which depends on the initial condition but for the stationary problem and one control-theoretic which does not depend on the initial condition but treats a class of interesting external forces.

Many options, another one:

$$\text{max total energy} + H_\infty \text{ norm} .$$

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It is well known that the best global damping matrix for a wide class of criteria is given by a **modal damping** matrix. These are matrices for which the vibrational system decomposes to a system of uncoupled 1 DOF systems.

For our mixed criterion, best global damping matrix is also modal damping matrix.

Numerical experiments confirm that mixed criteria lead to physically meaningful damping matrices.

# Thanks for the attention!

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