Interpolation-based parametric model reduction for efficient damping optimization

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Introduction Optimization problem



We consider vibrational system

$$M\ddot{q}(t) + \overbrace{(C_{int} + B_2 G B_2^T)}^{\text{C=damping part}} \dot{q}(t) + Kq(t) = E_2 w(t),$$
$$z(t) = H_1 q(t).$$

- M, K > 0 mass and stiffness,
- E_2 primary excitation matrix,
- C_{int} internal damping e.g. $C_{int} = \alpha_c C_{crit}$, where $C_{crit} = 2M^{1/2}\sqrt{M^{-1/2}KM^{-1/2}}M^{1/2}$,
- $G = \text{diag}(g_1, g_2, \dots, g_p), g_i \ge 0$ represents coefficients of damper,
- q state vector and z is output vector determined by H_1 ,
- vector w corresponds to primary excitation input.

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Introduction Optimization problem



Example: n-mass oscillator or oscillator ladder



 $M = \text{diag}(m_1, m_2, \dots, m_n), \quad C = B_2 G B_2^T + \alpha_c C_{crit},$ $B_2 G B_2^T = g_1 (e_k - e_{k+1}) (e_k - e_{k+1})^T + g_2 (e_j - e_{j+1}) (e_j - e_{j+1})^T.$



Introduction Optimization problem



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$$K = \begin{pmatrix} k_1 + k_2 & -k_2 & & \\ -k_2 & k_2 + k_3 & -k_3 & & \\ & \ddots & \ddots & \ddots & \\ & & -k_{n-1} & k_{n-1} + k_n & -k_n \\ & & & -k_n & k_n + k_{n+1} \end{pmatrix}$$

Introduction Optimization problem



Linearization

- Using Φ which simultaneously diagonalizes M and K

$$\Phi^T K \Phi = \Omega^2 = \mathrm{diag}(\omega_1^2, \dots, \omega_n^2) \quad \mathrm{and} \quad \Phi^T M \Phi = I,$$

$$0 < \omega_1 \leq \omega_2 \leq \ldots \leq \omega_n.$$

• with $\hat{x}_1 = \Omega \Phi^{-1} q(t)$ and $\hat{x}_2 = \Phi^{-1} \dot{q}(t)$ system can be written as :

$$\begin{split} \dot{\hat{x}}(t) &= \hat{A}\hat{x}(t) + \begin{bmatrix} 0 \\ \Phi^T E_2 \end{bmatrix} w(t), \\ z(t) &= \begin{bmatrix} H_1 \Phi \Omega^{-1} & 0 \end{bmatrix} x(t), \text{ where} \\ \hat{x} &= \begin{bmatrix} \hat{x}_1 \\ \hat{x}_2 \end{bmatrix}, \quad \hat{A} = \begin{bmatrix} 0 & \Omega \\ -\Omega & -\alpha\Omega - \Phi^T B_2 G B_2^T \Phi \end{bmatrix} \end{split}$$

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Introduction Optimization problem



Problem :

Determine "optimal" damping matrix ${\cal C}$ which will minimize the effect of the input w on the output z.

For criterion one can use e.g.:

• \mathcal{H}_2 norm of a system

$$\|H\|_{\mathcal{H}_2} = \left(\frac{1}{2\pi} \int_{-\infty}^{+\infty} \operatorname{tr} \left(H(j\omega)^* H(j\omega)\right) d\omega\right)^{\frac{1}{2}}$$

transfer function $H(s) = H_1(s^2M + sC + K)^{-1}E_2, \quad s \in \mathbb{C}.$

• \mathcal{H}_∞ norm of a system

$$\|H\|_{\mathcal{H}_{\infty}} = \sup_{\omega > 0} \|H(j\omega)\|.$$

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 $\label{eq:constant} \text{transfer function } H(s) = H_1(s^2M + sC + K)^{-1}E_2, \quad s \in \mathbb{C}.$

+ \mathcal{H}_∞ norm of a system

$$\|H\|_{\mathcal{H}_{\infty}} = \sup_{\omega \ge 0} \|H(j\omega)\|.$$

Introduction Optimization problem



Optimization problem for \mathcal{H}_2 norm

Impulse response energy leads to

$$\hat{A}^T \hat{X} + \hat{X} \hat{A} = -\hat{H}^T \hat{H},$$

$$\begin{split} \hat{A} &= \begin{bmatrix} 0 & \Omega \\ -\Omega & -\alpha \Omega - \Phi^T B_2 G B_2^T \Phi \end{bmatrix}, \quad \hat{H} &= \begin{bmatrix} H_1 \Phi \Omega^{-1} & 0 \end{bmatrix}, \\ \text{With } \hat{X} &= \begin{bmatrix} X_{11} & X_{12} \\ X_{21} & X_{22} \end{bmatrix}, \text{ it holds} \\ J_2 &= \operatorname{tr} \left(E_2^T \Phi X_{22} \Phi^T E_2 \right) \quad \to \min. \end{split}$$

Model order reduction: provides efficient approximation of energy J_2 needed for optimization of G.

Introduction Optimization problem



Optimization problem for \mathcal{H}_2 norm

Impulse response energy leads to

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With $\hat{X} = \begin{bmatrix} X_{11} & X_{12} \\ X_{21} & X_{22} \end{bmatrix}$, it holds
 $J_2 = \operatorname{tr} \left(E_2^T \Phi X_{22} \Phi^T E_2 \right) \quad \rightarrow \min.$

Model order reduction: provides efficient approximation of energy J_2 needed for optimization of G.

Model order reduction Interpolatory methods Damping optimization



Model order reduction

With $W_r \in \mathbb{R}^{n \times r}$, $q(t) = W_r q_r(t)$ and $V_r \in \mathbb{R}^{n \times r}$ we obtain reduced system

$$\begin{split} M_{r}\ddot{q}_{r}(t) + C_{r}\dot{q}_{r}(t) + K_{r}q_{r}(t) &= E_{r}w(t) \quad \text{where} \\ M_{r} &= W_{r}^{*}MV_{r}, C_{r} = W_{r}^{*}CV_{r}, \\ K_{r} &= W_{r}^{*}KV_{r}, E_{r} = W_{r}^{*}E_{2}, H_{r} = H_{1}V_{r} \end{split}$$

Transfer function of reduced system is

$$H_r(s) = H_r(s^2 M_r + sC_r + K_r)^{-1} E_r, \quad s \in \mathbb{C}.$$

We choose W_r and V_r to enforce (tangential) interpolation.

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Model order reduction Interpolatory methods Damping optimization



Model reduction by Interpolation

For selected points $\sigma_1, \sigma_2, \ldots, \sigma_r \in \mathbb{C}$ and directions b_1, \ldots, b_r and c_1, \ldots, c_r find $H_r(s)$ such that

$$c_i^T H(\sigma_i) = c_i^T H_r(\sigma_i)$$
$$H(\sigma_i)b_i = H_r(\sigma_i)b_i$$
$$c_i^T H'(\sigma_i)b_i = c_i^T H'_r(\sigma_i)b_i.$$

Moreover: we would like to have an approximation s.t. $\|\cdot\|_{\mathcal{H}_2}$ is optimally approximated, i.e. find local minimizer for $\|H - H_r\|_{\mathcal{H}_2}$.

This can be done in general framework efficiently¹ using structure preserving Model Reduction.

Additionally to preserve structure in second order system: $V_r=W_r.$

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Alg. 1: Iterative rational Krylov algorithm (IRKA)

Require: System matrices defining, and given gains g_1, g_2, \ldots, g_p , initial shift selection $\sigma_1 \ldots, \sigma_r$; initial tangent directions r_1, \ldots, r_r .

1:
$$V_r = [(\sigma_1^2 M + \sigma_1 C + K)^{-1} Br_1, \dots, (\sigma_r^2 M + \sigma_r C + K)^{-1} Br_r];$$

- 2: for $j=1,\ldots,\max_{it}$ do
- 3: Form reduced system determined by: $M_r = V_r^* M V_r$, $C_r = V_r^* C V_r$, $K_r = V_r^* K V_r$, $E_r = V_r^* E_2$, $H_r = H_1 V_r$
- 4: Consider quadratic eigenvalue problem $(M_r\lambda_i^2 + C_r\lambda_i + K_r)x_i = 0, x_i \neq 0, i = 1, \dots, 2r$ and reduce system to r states in order to have r eigenvalues $\tilde{\lambda}_1, \dots, \tilde{\lambda}_r$ closed under conjugation
- 5: $\sigma_i = \hat{\lambda}_i$ and update $r_i, i = 1, \dots, r$
- 6: $V_r = [(\sigma_1^2 M + \sigma_1 C + K)^{-1} Br_1, \dots, (\sigma_r^2 M + \sigma_r C + K)^{-1} Br_r];$
- 7: if converged then
- 8: return $V = V_r$
- 9: end if
- 10: end for

Model order reduction Interpolatory methods Damping optimization



Usage of modal coordinates

For given shift σ and direction r_i in solving

$$(\sigma^2 M + \sigma_1 C + K)^{-1} B r_i$$

we apply reduction directly to system in modal coordinates.

Here we use Sherman-Morrison-Woodbury formula:

$$(\sigma^{2}I + \sigma\alpha\Omega + \sigma\Phi^{T}B_{2}GB_{2}^{T}\Phi + \Omega^{2})^{-1}\Phi^{T}Br_{i} = T^{-1}\Phi^{T}Br_{i}$$
$$-sT^{-1}B_{g}(I_{p} + sB_{g}^{T}T^{-1}B_{g})^{-1}B_{g}^{T}T^{-1}\Phi^{T}Br_{i}$$

where

$$T = (\sigma^2 I + \sigma \alpha \Omega + \Omega^2),$$

$$B_g = \Phi^T B_2 \operatorname{diag}(\sqrt{g_1}, \dots, \sqrt{g_p}),$$

 $(I_p + s B_q^T T^{-1} B_g)$ has dimension p (p - number of dampers ($p \ll n$))

Model order reduction Interpolatory methods Damping optimization



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Model order reduction Interpolatory methods Damping optimization



Internal reduction

- a) internal reduction based on balanced truncation;
 used balanced truncation method applied to the linearized model.
- b) internal reduction based on IRKA algorithm; apply additional reduction using IRKA approach to linearized model.
- c) internal reduction based on dominant poles; Since the transfer function

$$F(s) = \sum_{i=1}^{2n} \frac{R_i}{s - \lambda_i} \quad \text{with} \quad R_i = (H_1 x_i) (y_i^* E_2) \lambda_i,$$

where $\lambda_i \in \mathbb{C}, x_i, y_i$ are, respectively, eigenvalues, right and left eigenvectors of the QEP. We maintain the r poles with the largest values of $\frac{||R_i||}{|\text{Re}(\lambda_i)|}$

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Model order reduction Interpolatory methods Damping optimization



- fixed sampling parameters (depending on parameter feasible area);
- adaptive sampling during optimization.



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Model order reduction Interpolatory methods Damping optimization



Optimization approach using fixed sampling parameters

Algorithm 2:

Require: System matrices; initial value for shift selection $\sigma_1 \dots, \sigma_r$ and initial directions r_1, \dots, r_r ;

number of wanted poles k_{wanted} for each setting of parameters.;

set of sampling parameters g^1, \ldots, g^m .

Ensure: Approximate optimal gains.

1: for
$$j=1,\ldots,m$$
 do

2: With IRKA Algorithm using gain g^i calculate V^i

3: end for

$$4: X = \operatorname{orth}([V^1, \dots, V^m]).$$

- 5: Form a reduced system using \boldsymbol{X} .
- 6: Find an optimal gains by using an appropriate optimization procedure on obtained reduced system.

Model order reduction Interpolatory methods Damping optimization



Optimization approach using adaptive sampling

Algorithm 3:

Require: System matrices; initial value for shift selection $\sigma_1 \dots, \sigma_r$ and initial directions r_1, \dots, r_r ;

number of wanted poles k_{wanted} for each setting of parameters.;

the first sampling parameters g^0 .

Ensure: Approximate optimal gains.

1:
$$j = 0;$$

2: repeat

- 3: With IRKA Algorithm using gain g^j calculate V^j
- 4: Form reduced system using $X = \operatorname{orth}([V^0, V^1, \dots, V^j])$.

5:
$$j = j + 1$$

- 6: Find an approximation of optimal gains by using obtained reduced system and denote it by g^j .
- 7: until $|g^j g^{j-1}| < tol_g$
- 8: return g^j

Model order reduction Interpolatory methods Damping optimization



Parametric dominant pole algorithm

Reduced system is obtained with $q(t) = Xq_k(t)$ where $X \in \mathbb{C}^{n \times k}$ span the eigenspaces associated with the k dominant poles, efficiently calculated.

For initial parameters $g^{(1)} = 0, g^{(2)}, \ldots, g^{(m)}$ we merge together corresponding right eigenspaces $X^{(j)}$. \rightsquigarrow for initial parameters the original and reduced model have similar behavior near dominant poles.

Determination of sampling parameters $g^{(2)}, \ldots, g^{(m)}$: adaptively, depending on residual error bound.

²P. Benner, P. Kürschner, Z. Tomljanović, N. Truhar, Semi-active damping optimization of vibrational systems using the parametric dominant pole algorithm, Journal of Applied Mathematics and Mechanics, 2016

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Example

Comparison with dominant pole algorithm Comparison of approaches based on IRKA



We will consider the n-mass oscillator with 1900 masses where

$$M = \operatorname{diag}(m_1, m_2, \dots, m_n),$$

$$K = \begin{pmatrix} 2k_1 + 2k_2 & -k_2 & -k_3 & & \\ -k_2 & 2k_2 + 2k_3 & -k_3 & -k_4 & & \\ -k_3 & -k_3 & 2k_3 + 2k_4 & -k_4 & -k_5 & \\ & \ddots & \ddots & \ddots & \ddots & \ddots & \end{pmatrix}.$$

With configuration:

$$n = 1900; \quad \alpha_c = 0.005$$

$$k_i = 500, \quad \forall i; \qquad m_i = \begin{cases} 144 - \frac{3}{20}i, & i = 1, \dots, 475, \\ \frac{i}{10} + 25, & i = 476, \dots, 1900. \end{cases}$$

Example Comparison with dominant pole algorithm Comparison of approaches based on IRKA



Primary excitation matrix is applied to 10 consecutive masses, i.e.

 $E_2(471:481,1:10) = \operatorname{diag}(10,20,30,40,50,50,40,30,20,10),$

We are interested in the 18 states equally distributed

 $H_1(1:18,100:100:1800) = I_{18\times 18}$

The geometry of external damping is determined by four dampers with

$$B_2 = [e_i \ e_{i+1} \ e_k \ e_{k+1}],$$

for 44 different damping configurations (i, k), where

 $G = \operatorname{diag}(g_1, g_1, g_2, g_2).$

Comparison:

- IRKA with fixed sampling \leftrightarrow
- IRKA with fixed sampling $\ \leftrightarrow$

parametric dominant pole algorithm, IRKA with adaptive sampling.

Comparison with dominant pole algorithm

Comparison of approaches based on IRKA

Example





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Gains for fixed sampling

$$g^{(0)} = (0, 0) \implies g^{(1)}$$
$$g^{(2)} = (1000, 1000)$$
$$g^{(3)} = (100, 1000)$$
$$g^{(4)} = (1000, 100)$$



Example

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Comparison with dominant pole algorithm Comparison of approaches based on IRKA



Comparison of relative errors in optimal gain



Example Comparison with dominant pole algorithm Comparison of approaches based on IRKA



Example

Comparison with dominant pole algorithm Comparison of approaches based on IRKA



Shank you for your attention!